



Modelling of transport phenomena in gas tungsten arc welding

A. Farzadi*, S. Serajzadeh, A.H. Kokabi

Materials Science and Engineering Department, Sharif University of Technology,
Azadi Ave., Tehran, Iran

* Corresponding author: E-mail address: farzadi@mehr.sharif.edu

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ABSTRACT

Purpose: Since numerical heat transfer and fluid flow models have provided significant insight into welding process and welded materials that could not been achieved otherwise, there has been an important interest in the quantitative representation of transport phenomena in the weld pool. On the other hand, the temperature and velocity distributions of the molten metal as well as the cooling rate after welding operation affect the weld geometry, the microstructure, and the mechanical properties of weld zone. This work demonstrates that the application of numerical transport phenomena can significantly add to the quantitative knowledge in welding and help the welding community in solving practical problems.

Design/methodology/approach: The temperature and velocity fields are simulated using the solution of the equations of conservation of mass, energy and momentum in three-dimension and under steady-state heat transfer and fluid flow conditions.

Findings: The weld pool geometry and various solidification parameters were calculated. The calculated weld pool geometries were in good agreement with the ones obtained using the experiments. The solidification parameters of G and G/R are determined. It is found that as the welding speed increases, the value of G/R at the weld pool centerline decreases.

Research limitations/implications: Welding process used in this study is gas tungsten arc (GTA) welding and base metal is commercial pure aluminum. This model can be investigated to simulate other materials and welding processes. Also the results of this study such as the temperature field can be used in the simulation of microstructure, mechanical properties, etc of welding zone.

Originality/value: In this research the solidification parameters of G , R and G/R can be used for prediction of the solidification morphology and the scale of the solidification structure.

Keywords: Computational mechanics; GTA welding; Transport phenomena; Commercial pure aluminium

METHODOLOGY OF RESEARCH, ANALYSIS AND MODELLING

1. Introduction

In first attempts, significant progress has been made in the solution of the equations of conservation of mass, momentum and energy in fusion welding with a stationary heat source [1]. However, in practice, the heat source moves with a certain velocity. Thus, several investigations of fluid flow and heat

transfer in welding with moving heat source have been reported [2, 3]. The three dimensional (3D) comprehensive numerical heat transfer and fluid flow model used in the calculations has been extensively validated for the welding of steels [4-7], aluminum alloys (5182 and Al-Cu) [8, 9], and titanium alloys [10]. However, most of these studies were concerned with iron and its alloys while the heat transfer and fluid flow in the commercial pure aluminum have not received much attention in the literature.

Numerical calculations of heat transfer and fluid flow in welding have enabled accurate quantitative calculations of thermal cycles and fusion zone geometry [2, 11]. Capabilities to quantitatively understand geometry, composition and structure of welds in simple systems have provided hope that one day welding engineers may be able to use numerical models to tailor weldment characteristics according to specifications. In reality, the numerical heat transfer and fluid flow codes for fusion welding have so far been used mostly as a research tool [4, 5] rather than as a tool for design and manufacturing in the industry but application of commercial code such as FLUENT can aid to reach this goal.

In this study, the heat transfer and fluid flow GTA welds of commercial pure aluminum were simulated using a well-tested, three-dimensional, steady state, numerical heat transfer and fluid flow model. The model was applied to understand the temperature field and the weld dimensions for various GTA welding conditions. For the accuracy of the calculation, a very fine grid system was used. Verification of the model was performed through comparing the calculated results with metallographic weld cross sections. Several important solidification parameters, including solidification rate (R) and temperature gradient (G) at the weld centerline, were calculated. These parameters may be used for determination of the solidification morphology and the scale of the solidification substructure.

2. Mathematical formulation

An incompressible laminar and Newtonian liquid flow is assumed in the weld pool. The density variation in the calculation domain is ignored except for the calculation of the buoyancy force following Boussinesq's approximation. Electromagnetic, surface tension, and buoyancy driving forces were considered for the calculation of weld pool convection. The weld pool boundary was traced using an enthalpy-porosity technique in a fixed Cartesian coordinate system.

2.1. Governing equations

Using the assumptions stated above, the circulation of liquid metal in the weld pool can be represented by the following linear momentum conservation equation for the j th direction:

$$\rho \frac{\partial(u_i u_j)}{\partial x_i} = -\frac{\partial P}{\partial x_j} + \frac{\partial}{\partial x_i} \left(\mu \frac{\partial u_j}{\partial x_i} \right) + \frac{\partial}{\partial x_j} \left(\mu \frac{\partial u_i}{\partial x_j} \right) + \rho U \frac{\partial u_i}{\partial x_i} - C \left(\frac{(1-f_L)^2}{f_L^3 + B} \right) u_j + S b_j + S e_j \quad (1)$$

where ρ is density, x_i is the distance along the $i=1, 2,$ and 3 directions, u_j is the velocity component along the j direction, μ is the viscosity, p is the pressure, f_L is the liquid fraction, B is a constant introduced to avoid division by zero, C is a constant that considers mushy zone morphology, U is the welding speed along the positive x direction, and $S b_j$ and $S e_j$ present the electromagnetic and buoyancy source terms, respectively. In Eq. (1), the first term on the right-hand side is the pressure gradient. The second term is the viscosity arising from casting the momentum equation into a general form. The fifth term describes the frictional dissipation in the mushy zone according to the

Carman-Kozeny equation for flow through a porous media. The calculation of the electromagnetic and buoyancy source terms is expressed in the literatures [2, 12].

The pressure field was obtained by solving following continuity equation simultaneously with the momentum equation:

$$\frac{\partial(\rho u_i)}{\partial x_i} = 0 \quad (2)$$

The total enthalpy H is represented by a sum of sensible heat h and latent heat content ΔH , i.e., $H = h + \Delta H$, where $h = \int C_p dT$, C_p is the specific heat, T is the temperature, $\Delta H = f_L L$, L is the latent heat of fusion, and the liquid fraction f_L is assumed to vary linearly with temperature in the mushy zone:

$$f_L = \begin{cases} 1, & T > T_L \\ \frac{T - T_S}{T_L - T_S}, & T_S \leq T \leq T_L \\ 0, & T < T_S \end{cases} \quad (3)$$

where T_L and T_S are the liquidus and the solidus temperatures, respectively. The thermal energy transport in the weld workpiece can be expressed by the following modified energy equation:

$$\rho \frac{\partial(u_i h)}{\partial x_i} = \frac{\partial}{\partial x_i} \left(\frac{k}{C_p} \frac{\partial h}{\partial x_i} \right) - \rho \frac{\partial(u_i \Delta H)}{\partial x_i} + \rho U \frac{\partial \Delta H}{\partial x_i} + \rho U \frac{\partial h}{\partial x_i} \quad (4)$$

where k is the thermal conductivity.

2.2. Solution procedure

A 3D Cartesian coordinate system is used in the calculation, while only half of the workpiece is considered since the weld is symmetrical about the weld center line. These boundary conditions are further discussed as follows.

The top surface is assumed to be flat and subject to shear stress due to Marangoni force. Also u_z is defined to be zero at the surface. The heat flux on the top surface is given as

$$k \frac{\partial T}{\partial x_3} \Big|_{top} = \frac{f Q \eta}{r_b^2} \exp\left(-\frac{f(x^2 + y^2)}{r_b^2}\right) - h(T - T_a) \quad (5)$$

where r_b is the heat distribution parameter, f is the power distribution factor, Q is the total arc power, η is the power efficiency, h is the heat transfer coefficient, and T_a is the ambient temperature.

At the symmetric surface, u_x and the normal gradients of u_x and u_z are taken as zero. In addition, the gradient of enthalpy, dh/dy , is zero.

At all other surfaces, the heat losses by convection and the velocities are set to be zero.

The detailed method of discretizing the governing equations is available in the literature [2, 15]. The governing equations are discretized using the control volume method [13], where the computational domain is divided into small rectangular control volumes. A staggered grid is used to store the velocity

components. To numerically solve the governing equations with the associated source terms, a general computational fluid dynamics (CFD) computer program, FLUENT code was used. The material properties employed in the calculation of fluid flow and heat transfer are summarized in Table 1 [4]. The velocity-pressure coupling in the discretized momentum equations is handled using SIMPLE algorithm at the each iteration [15, 16].

Accurate calculation of weld temperature and velocity fields, and the temperature gradients requires the use of a very fine grid system. A typical grid system used in the present study contained 143x73x50 grid points, and the corresponding computational domain had dimensions of 100 mm in length, 45 mm in width, and 5 mm in depth. Spatially non uniform grids were used for maximum resolution of variables. A finer grid spacing was used near the heat source. The minimum grid spacings along the x and z directions were about 50 and 20 μ m, respectively.

In the present study, the convergence was reached when $R \leq 10^{-4}$ for the energy equation and $R \leq 10^{-3}$ for the momentum equations.

Table 1.

Physical properties of commercial pure aluminum used in the calculations [14]

Physical Property	Value
Liquid temperature, T_L (K)	930
Solid temperature, T_S (K)	916
Density of liquid metal, ρ (kg/m ³)	2700
Viscosity of liquid, μ (kg/m s)	0.0014
Thermal conductivity of solid, k_S (J/m s K)	222
Thermal conductivity of liquid, k_L (J/m s K)	108
Specific heat of solid, C_{PS} (J/kg K)	904
Specific heat of liquid, C_{PL} (J/kg K)	904
Latent heat of melting, ΔH (J/kg)	3.95×10^5
Temperature coefficient of surface tension, $d\gamma/dT$ (N/m K)	-0.152×10^{-3}
Coefficient of thermal expansion, β (K ⁻¹)	2.55×10^{-5}
Magnetic permeability, μ_m (N/A ²)	1.26×10^{-6}

2.3. Experimental procedure

Commercial pure aluminum plate with the thickness of 5 mm was used in this investigation. The composition of the as-received bar was (by wt%): 0.08 Si, 0.29 Fe, 0.005 Cu, 0.002 Mn, 0.002 Mg, 0.06 Zn, 0.003 Ti, 0.001 Cr, and 0.002 Ni. The welding electrode was made of W-2%Th measuring 2.4 mm in diameter. High Purity (99.99%) argon was used as welding gas. In welding experiments, the arc current and welding speed were 110 A, 15 V and 2.7 mm/s, respectively.

3. Results and discussion

3.1. Temperature and velocity fields

The experimentally determined weld pool cross sections are compared with the corresponding calculated geometries in Figure 1. It is observed that the computed weld pool geometry and dimensions

agree well with the experimental results. In these figures, the mushy zone, i.e. the solid and liquid two phase region is presented with the distinct color. In the present study, the mushy zone is determined as the region having temperature between the solidus (916 K) and liquidus (930 K) isotherms.

Figure 2 shows the computed 3D temperature and velocity fields for a weld pool made by welding conditions presented in section 2.3. For clarity, only a half of weld is shown. The liquid metal motion in the weld pool is driven mainly by the surface tension and electromagnetic forces, and to a much lesser extent, by the buoyancy force. The relative importance of these driving forces is quantitatively examined in a subsequent section. Because the temperature coefficient of surface tension $d\gamma/dT$ has a negative value, the surface tension force drives the liquid metal from the center to the periphery at the top surface of the weld pool. The peak temperature and maximum velocity are 1746 K and 1.445 m/s, respectively.

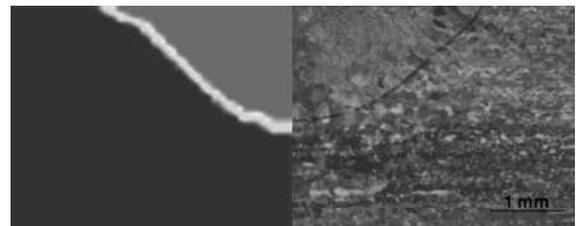


Fig. 1. Experimental and calculated weld pool cross sections for GTA welds. Welding condition was expressed in Sec. 2.3

3.2. Solidification

The average temperature gradient, \overline{G}_a , of the weld pool along a given plane is defined as

$$\overline{G}_a = \frac{T_P - T_S}{d} \quad (6)$$

where T_P is the peak temperature, T_S is the solidus temperature, and d is the distance between the location of peak temperature and boundary at the weld pool centerline.

The temperature gradient and the solidification rate are important in the combined forms G/R as they influence the solidification morphology and the scale of the solidification substructure, respectively. When the value of G/R increases, the interface morphology changes from equiaxed-dendritic, to cellular-dendritic, to cellular grains [17]. Figure 3 shows the temperature gradient G and G/R at the weld centerline for different welding speed. It is shown that with the increase in welding speed the temperature gradient decreases and the value of G/R decreases at the weld pool centerline. Thus the solidification structure of fusion zone would be affected by welding speed.

4. Conclusions

The temperature and velocity fields, weld pool size, and several solidification parameters during GTA welding of commercial pure aluminum were simulated using a comprehensive three-dimensional

heat transfer and fluid flow model. The major conclusions can be made from investigation are as follows:

- (1) The geometry of fusion zone predicted from the three-dimensional steady state heat transfer and fluid flow model using FLUENT code are in a good agreement with the corresponding experimental results.
- (2) The value of G/R at the weld pool centerline is shown to decrease with the increase in welding speed.

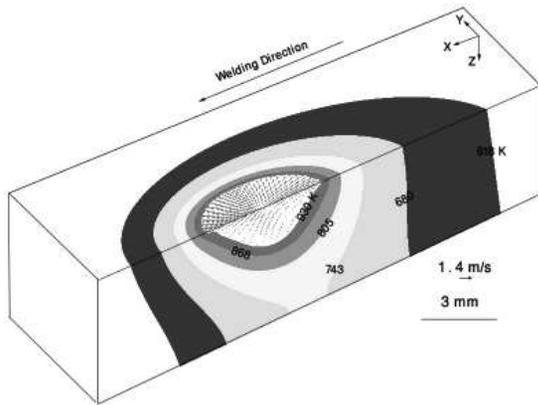


Fig. 2. Calculated temperature and velocity fields in three dimensions. Welding condition was presented in Sec. 2.3

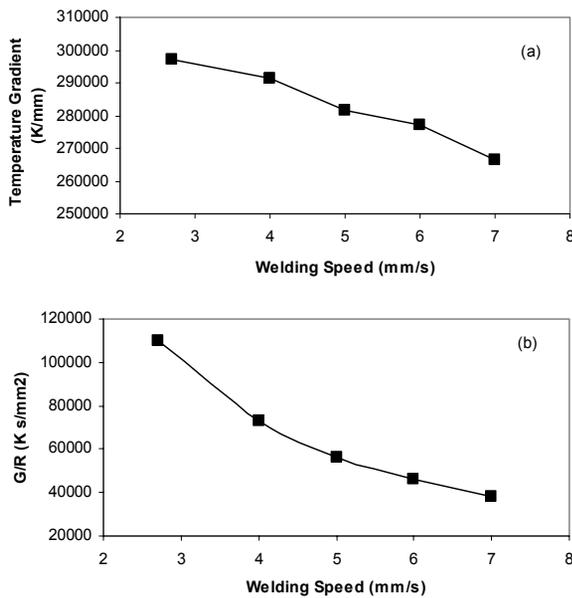


Fig. 3. Calculated values of (a) G , (b) GR and (c) G/R at the weld pool centerline at different welding speed

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